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Dispersion Characteristics of SH waves in a Rotating Micro-structured Plate with Restrained Boundaries

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Abstract

Shear horizontal (SH) waves in plates are elastic stress disturbances that propagate along bounded surfaces, with in-plane shear motion perpendicular to the propagation direction, influenced by physical, geometric, and boundary conditions. Due to their sensitivity to surface and subsurface defects, SH waves are widely employed in non-destructive testing (NDT), structural health monitoring (SHM), and material characterization. This study presents a theoretical investigation of SH-wave propagation in a rotating, microstructural elastic plate within the framework of Consistent Couple Stress Theory (CCST), incorporating elastically restrained boundary conditions. CCST introduces a characteristic length parameter to account for microstructural and size-dependent effects beyond classical elasticity. Restrained boundaries provide an intermediate model between idealized traction-free and rigid conditions, capturing realistic boundary behavior encountered in practical applications. An analytical dispersion relation is derived to examine the combined effects of boundary restraints, microstructural parameters, and rotational speed on SH-wave dispersion, with several special cases presented to validate the general formulation. Graphical results illustrate the influence of restraint stiffness, characteristic length, plate thickness, and rotation on SH-wave dispersion characteristics. The findings may offer valuable insights for the design of advanced sensing devices, the development of non-destructive evaluation techniques, and the analysis of seismic wave propagation in rotating geophysical layers.

Keywords: SH waves, Couple stress theory, Stiffness coefficients, Restrained boundary conditions, Dispersion curves, Rotating elastic plate.

1 Introduction

Waves propagating in rotating elastic plates have garnered considerable attention due to their relevance in both engineering and geophysical domains. Rotation introduces coriolis and centrifugal effects, which significantly alter the characteristics of wave motion—the coriolis force modifies wave trajectories and dispersion, while the centrifugal force shapes the equilibrium state. Understanding these influences may aid the design, analysis, and health monitoring of rotating systems in various engineering applications, where stress waves and vibrations can affect performance and reliability. In broader contexts, wave propagation in rotating layers may offer insights into seismic wave behavior, earthquake mechanics, and planetary dynamics and may also support the development of advanced sensing and non-destructive evaluation techniques. BHUTA and JONES (1963) established the foundational theoretical framework for analyzing the influence of uniform rotation on the propagation characteristics of elastic waves. Their study was the first to reveal the occurrence of elliptical polarization in wave motion, arising from the coriolis-induced coupling between displacement components. Further, Schoenberg and Censor explored plane wave solutions in rotating elastic solids in two (CENSOR, SCHONEBERG, 1973) and three-dimensions (SCHONEBERG, CENSOR, 1973). They have found that these waves do not exhibit purely shear or purely compressional characteristics and showed that the wave speeds are governed by the ratio of the rotational frequency of the medium to the angular frequency of the waves. Extensive studies have been conducted and documented in the literature (ADVANI, 1967; ADVANI, BULKELEY, 1969; BULKELEY, 1973; HASHEMI, RICHARD, 2001; HUANG, WANG, 2001; LAMB, SOUTHWELL, 1921; LUO, MOTE JR., 2000; NOWINSKI, 1981; SOUTHWELL, 1922; AURIAULT, 2004; KUMAR, CHAWLA, 2011; YU, YU, 1996) on wave propagation in different rotating elastic medias such as beams, disks, membranes, half-spaces and plates. Collectively, the findings demonstrate that rotation introduces strong coupling between different wave modes, profoundly influencing the stability, dispersion behavior, and dynamic response of the system. These insights significantly advance the understanding of wave-rotation interactions and the complex challenges associated with the analysis and design of rotating machinery.

Shear horizontal (SH) waves represent a fundamental class of elastic wave motion with broad applicability across geophysics, material science, structural health monitoring (SHM), and ultrasonic diagnostics. SH waves exhibit pure in-plane transverse particle motion perpendicular to the propagation direction, rendering them immune to mode conversion at free or bonded interfaces. This distinctive feature imparts high signal clarity, strong penetration through layered structures, and minimal dispersion, making SH waves exceptionally effective for non-destructive testing (NDT), surface acoustic wave (SAW) sensors and characterization of multilayered or anisotropic materials. When propagating in thin plates, SH waves are confined and guided by the plate boundaries, resulting in well-defined dispersion characteristics and enhanced sensitivity to surface, subsurface, and interfacial discontinuities. CASTING and HOSTEN (2001) investigated the interaction between non-principal planes and the surrounding fluid in the propagation of quasi-shear horizontal (quasi-SH) modes, demonstrating that this coupling

73 enables the efficient generation and detection of such modes using air-coupled transducers in
74 solid plates. Subsequently, LEE *et al.* (2010) introduced a novel technique for damage detection
75 in plates based on focused SH waves, which significantly minimizes the number of transducers
76 required while enhancing detection accuracy. KUZNETSOV (2022) investigated the behavior of
77 SH waves in functionally graded plates, incorporating bifurcation effects to enhance the under-
78 standing of their dynamic response and potential engineering applications. Shear horizontal
79 (SH) waves in thin plates find extensive applications across various branches of engineering,
80 serving numerous practical purposes such as material characterization, design and optimiza-
81 tion of multilayered composite structures, and damping analysis in automotive, aerospace, and
82 research industries (KUZNETSOV, 2006; PHAN *et al.*, 2019; DJERAN-MAIGRE, KUZNETSOV,
83 2014; SIMONETTI, CAWLEY, 2004; JIANGONG, 2011).

84 In practical engineering applications, plates are often influenced by interactions with their
85 surrounding environments and boundary surfaces, which significantly affect their dynamic and
86 mechanical responses. However, capturing these complex interactions with full physical ac-
87 curacy is often difficult; hence, simplified boundary idealizations are introduced to make the
88 analysis tractable. The most common boundary conditions include the stress-free boundary,
89 representing surfaces devoid of external constraints and the fixed (clamped) boundary, corre-
90 sponding to fully restrained conditions where both displacement and rotation are restricted. To
91 address the practical scenarios that lie between the two extreme boundary conditions, MINDLIN
92 (1960) introduced the concept of elastically restrained boundary conditions (ERBC) in the nor-
93 mal direction, modeling the boundary as exhibiting spring-like behavior governed by Hooke's
94 law. Subsequently, GRAFF (2012) elaborated on these conditions and referred to them as
95 mixed boundary conditions, which provide a continuous transition between stress-free and
96 rigidly fixed boundaries. MOUKHOMODIAROV *et al.* (2010) further examined this transition
97 in a linear isotropic elastic layer, establishing a framework for intermediate boundary model-
98 ing. Expanding upon this foundation, MOUKHOMODIAROV and ROGERSON (2012a, 2012b)
99 employed various analytical approaches to investigate the influence of ERBC on long-wave mo-
100 tion, and later extended their study to analyze long-wave dispersion phenomena in pre-stressed,
101 incompressible elastic layers under ERBC (MOUKHOMODIAROV, ROGERSON, 2013). More re-
102 cently, PAIMUSHIN and GAZIZULLIN (2018) explored the effect of ERBC by introducing elastic
103 spacers connecting the plate to a rigid frame, while KUZNETSOV (2015) analyzed the prop-
104 agation characteristics of Lamb waves in elastic layers constrained by clamped and partially
105 clamped boundaries.

106 Wave propagation is governed not only by the geometry and boundary conditions of the
107 structure but also by the intrinsic material properties. In essence, the propagation behav-
108 ior—depending on the medium and the nature of the wave is determined by the complex inter-
109 play among material characteristics, structural geometry, and boundary constraints. Although
110 the behavior of shear-horizontal (SH) waves has been extensively studied in the context of ro-
111 tational effects, it has not yet been comprehensively investigated within the framework of con-
112 sistent couple stress theory (CCST). HADJESFANDIARI and DARGUSH (2011) developed CCST

113 following the work of MINDLIN and TIERSTEN (1962) and KOITER (1969) generalising and
114 extending the classical theory of elasticity. It addresses the limitations of earlier couple stress
115 theories (MINDLIN, TIERSTEN, 1962; KOITER, 1969; TOUPIN, 1962; VOIGT, 1887; COSSERAT,
116 COSSERAT, 1970; YANG *et al.*, 2002) by introducing skew-symmetric couple stress tensor and
117 refined couple stress coefficient ($\eta = \mu l^2$) determined by bulk modulus (μ) and characteristic
118 length of the material (l). Skew-symmetric couple stress tensor captures internal rotations and
119 moment stresses while characteristic length scale parameter enables to study size-dependent
120 effects and microstructural interactions. Development of size-dependent couple stress theory in
121 Timoshenko beams results in an increased stiffness predictions compared to classical beam the-
122 ories, highlighting the importance of size effects in structural analysis (ASGHARI *et al.*, 2011).
123 DARGUSH, *et al.* (2021) conducted two-and three-dimensional elastostatic analysis applying
124 Ritz variational approach within the framework of size-dependent couple stress theory.

125 Several studies have also examined wave propagation in micro-structured plates using CCST
126 models, focusing primarily on size-dependent effects and couple-stress elasticity. GHODRATI
127 *et al* (2018) applied CCST to study the lamb waves propagation considering microstructure
128 effects in very high frequencies. HUANG *et al.* (2024) analysed Lamb waves in micro/nano-
129 plates subjected to CCST . Their findings show that in anti-symmetric modes, the plate surface
130 exhibits no in-plane displacement while at specific frequencies on the intersection points of
131 symmetric and anti-symmetric dispersion curves out-of-plane displacement vanishes on the
132 free surface. WU and HSU (2022) observed the mechanical behavior of functionally graded
133 microplate accounting for size effects using the consistent couple stress theory. SHARMA and
134 KUMAR (2015) not only studied the lamb waves in plate with the liquid loadings but also
135 investigated the effects of rotation and thermal parameters on both plane and Rayleigh waves
136 using CCST (SHARMA, KUMAR, 2023). WU and HU, (2021) proposed a unified size-dependent
137 plate theory for static bending and free vibration analyses of micro- and nano-scale plates using
138 the consistent couple stress theory.

139 The consistent couple stress theory (CCST) has been extensively utilized in various struc-
140 tural analyses (WANG *et al.*, 2017; KAUR *et al.*, 2024; DEEP, SHARMA, 2023; SHARMA, KUMAR,
141 2014), its application to shear-horizontal (SH) wave propagation in plates remains a relatively
142 unexplored domain. The present study introduces a novel investigation into the propagation be-
143 havior of SH waves in a rotating microstructural plate subjected to elastically restrained bound-
144 ary conditions (ERBC). The formulation employs CCST, which incorporates a characteristic
145 length parameter linking the macroscopic response of the material to its intrinsic microstructure,
146 thereby enhancing accuracy for micro- and meso-scale applications. Furthermore, the inclusion
147 of additional terms in the equations of motion, corresponding to centripetal and coriolis effects,
148 enables a more realistic representation of rotational influences. Elastically restrained boundary
149 conditions, which interpolate between fully stress-free and fully clamped states, are critical for
150 accurately modeling the behavior of plates in practical engineering systems. Comprehensive
151 graphical analyses are presented to illustrate the effects of microstructural parameters, char-
152 acteristic length, plate thickness, and boundary conditions on phase velocity under rotational

153 motion. The outcomes are compared with non-rotating cases, providing useful insights into
 154 the influence of rotation on wave behavior. These findings may have potential relevance to a
 155 broad range of engineering applications, particularly in contexts where size-dependent effects
 156 and boundary constraints are significant.

157 2 Geometry and basic equations

158 Consider a thin micro-structured plate that is unbounded in the x - and z - directions and
 159 possesses a finite thickness of $2b$ along the y - axis ($-\infty < x < \infty$, $-b \leq y \leq b$, $-\infty < z < \infty$).
 160 The Cartesian coordinate system has an origin O located at the geometric center of the plate,
 161 equidistant from its upper and lower boundaries. The positive y - direction is taken vertically
 162 upward, through the thickness of the plate. The free surfaces of the plate are defined by the
 163 planes $y = b$ and $y = -b$. The medium (plate) is assumed to be rotating uniformly about an
 164 axis in the $x - y$ plane with angular velocity $\vec{\Omega} = (\Omega_1, \Omega_2, \Omega_3)$ and wave propagation is taken
 165 to travel along the x -direction.

166 For the two dimensional solution, all the physical quantities are supposed to be invariant
 167 along the z -direction, i.e., $\frac{\partial}{\partial z} \equiv 0$. The displacement vector \vec{u} is expressed as $\vec{u} = (u(x, y, t),$
 168 $v(x, y, t), w(x, y, t))$, where u, v and w denotes the displacement components in the x, y and z
 169 directions respectively and t denotes time. According to the consistent couple stress theory, the
 170 equation of motion of a rotating body in the absence of body forces is given as (SCHOENBERG,
 171 CENSOR, 1973; HADJESFANDIARI, DARGUSH, 2011)

$$(\lambda + \mu + \eta \nabla^2) \vec{\nabla}(\vec{\nabla} \cdot \vec{u}) + (\mu - \eta \nabla^2) \nabla^2 \vec{u} = \rho \left[\frac{\partial^2 \vec{u}}{\partial t^2} + (\vec{\Omega} \times (\vec{\Omega} \times \vec{u})) + \left(2\vec{\Omega} \times \frac{\partial \vec{u}}{\partial t} \right) \right] \quad (1)$$

172 where $\vec{u} = (u, v, w)$ is displacement vector, λ and μ are Lamé's constants, $\eta = \mu l^2$ is couple
 173 stress coefficient, l is characteristic length, ρ is density of material, $\vec{\Omega}$ is rotation vector and
 174 $\nabla^2 = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2}$. The term $\vec{\Omega} \times (\vec{\Omega} \times \vec{u})$ represents the centripetal acceleration resulting
 175 from time varying motion and the coriolis acceleration is denoted by the term $2\vec{\Omega} \times \frac{\partial \vec{u}}{\partial t}$.

176 The constitutive relations for consistent couple stress theory are given as

$$\sigma_{ji} = \lambda u_{k,k} \delta_{ij} + \mu (u_{i,j} + u_{j,i}) - \eta \nabla^2 (u_{i,j} - u_{j,i}) \quad (2)$$

$$\mu_{ji} = 4\eta (\bar{\omega}_{i,j} - \bar{\omega}_{j,i}), \text{ where } \bar{\omega}_i = \frac{1}{2} \epsilon_{ijk} u_{k,j} \quad (3)$$

177 where, σ_{ji} is force stress tensor, μ_{ji} is couple stress tensor, δ_{ij} is Kronecker's delta, ϵ_{ijk} is
 178 permutation tensor, $\bar{\omega}_i$ is rotation vector and $i, j, k = 1, 2, 3$.

3 Propagation of Shear Horizontal waves under rotation

Consider SH waves, propagating in a micro-structural plate under rotation. For the two-dimensional plate in the $x-y$ plane, the components of displacement and rotation are considered as $\vec{u} = (0, 0, w)$ and $\vec{\Omega} = (\Omega_1, \Omega_2, 0)$ respectively. Eq. (1) is transformed into the single partial differential equation as written below

$$C_2^2(1 - l^2\nabla^2)\nabla^2 w = \frac{\partial^2 w}{\partial t^2} - w(\Omega_1^2 + \Omega_2^2), \quad \text{where } C_2 = \sqrt{\frac{\mu}{\rho}} \quad (4)$$

let the solution of Eq. (4) is assumed as

$$w = h(y)e^{i\xi(x-Ct)} \quad (5)$$

where $h(y)$ is the unknown function of y , ξ is a wave number, C is phase speed, $i = \sqrt{-1}$. On substituting Eq. (5) into (4), Eq. (4) reduces to the following fourth-order differential equations.

$$\frac{d^4 h(y)}{dy^4} - \left(\frac{1}{l^2} + 2\xi^2\right) \frac{d^2 h(y)}{dy^2} + \left(\xi^4 + \frac{\xi^2}{l^2} \left(1 - \frac{C^2}{C_2^2}\right) - \left(\frac{\Omega_1^2 + \Omega_2^2}{C_2^2 l^2}\right)\right) h(y) = 0 \quad (6)$$

Solution of the above differential Eq. (6) is obtained as

$$h(y) = D_1 \text{Cosh}(R_1 y) + D_2 \text{Sinh}(R_1 y) + D_3 \text{Cos}(R_2 y) + D_4 \text{Sin}(R_2 y) \quad (7)$$

where D_1, D_2, D_3, D_4 are arbitrary constants and

$$R_1 = \sqrt{\xi^2 + \frac{1}{2l^2} \left(1 + \sqrt{1 + \frac{4\xi^2 C^2 l^2}{C_2^2} + \frac{4l^2}{C_2^2} (\Omega_1^2 + \Omega_2^2)}\right)}$$

$$R_2 = \sqrt{\frac{1}{2l^2} \left(\sqrt{1 + \frac{4\xi^2 C^2 l^2}{C_2^2} + \frac{4l^2}{C_2^2} (\Omega_1^2 + \Omega_2^2)} - 1\right) - \xi^2}$$

Thus, displacement component in Eq. (5) becomes

$$w(y) = \left(D_1 \text{Cosh}(R_1 y) + D_2 \text{Sinh}(R_1 y) + D_3 \text{Cos}(R_2 y) + D_4 \text{Sin}(R_2 y) \right) e^{i\xi(x-Ct)} \quad (8)$$

191 The non-zero component of stress and couple stress tensors are obtained as

$$\begin{aligned} \sigma_{yz} &= \mu \frac{\partial w}{\partial y} - \mu l^2 \left(\frac{\partial^3 w}{\partial x^2 \partial y} + \frac{\partial^3 w}{\partial y^3} \right) \\ &= \left(D_1 L_1 \text{Sinh}(R_1 y) + D_2 L_1 \text{Cosh}(R_1 y) - D_3 L_2 \text{Sin}(R_2 y) + D_4 L_2 \text{Cos}(R_2 y) \right) e^{i\xi(x-Ct)} \end{aligned} \quad (9)$$

$$\begin{aligned} \mu_{yx} &= 2\mu l^2 \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} \right) \\ &= \left(D_1 L_3 \text{Cosh}(R_1 y) + D_2 L_3 \text{Sinh}(R_1 y) - D_3 L_4 \text{Cos}(R_2 y) - D_4 L_4 \text{Sin}(R_2 y) \right) e^{i\xi(x-Ct)} \end{aligned} \quad (10)$$

192 where $L_1 = \mu R_1 + \mu l^2 \xi^2 R_1 - \mu l^2 R_1^3$, $L_2 = \mu R_2 + \mu l^2 \xi^2 R_2 + \mu l^2 R_2^3$, $L_3 = 2\mu l^2 (R_1^2 - \xi^2)$, $L_4 =$
193 $2\mu l^2 (R_2^2 + \xi^2)$.

194 4 Restrained boundary conditions

195 Elastically restrained boundary conditions (ERBC) can be conceptualized as boundary
196 surfaces embedded with distributed springs. These springs simulate elastic restraints by im-
197 posing resistance to displacements and rotations. To incorporate restrained boundary condi-
198 tions within a couple stress elastic model, it is necessary to account not only for the classical
199 stress-strain relations but also for additional stiffness coefficients and couple stress effects.
200 These coefficients govern the relationship between force stresses and displacements, as well as
201 between couple stresses and rotation gradients. Consequently, such conditions permit limited
202 displacement and rotation at the boundaries. Shear spring constant (k_1) representing resis-
203 tance to tangential (shear) deformation while (k_2) represents the rotational spring constant,
204 indicating resistance to rotational motion at the boundary. Here, *rotation* refers to the angular
205 motion of the plate itself, and its influence is incorporated into the dispersion relation of shear-
206 horizontal (SH) waves by accounting for coriolis and centrifugal forces within the framework of
207 rotating couple stress theory. For SH wave propagation, the tangential and rotational stiffness
208 coefficients respectively characterize the boundary's resistance to tangential displacement and
209 rotational deformation. Boundary conditions at the two surfaces ($y = \pm b$) of the plate are as
210 follows (MINDLIN, 1960; WANG *et al.*, 2017).

211 1) $\sigma_{yz} = k_1 w$

212 This boundary condition leads to following equation in four unknown arbitrary constants
213 $D_1 L_1 \text{Sinh}(R_1 y) + D_2 L_1 \text{Cosh}(R_1 y) - D_3 L_2 \text{Sin}(R_2 y) + D_4 L_2 \text{Cos}(R_2 y) - k_1 D_1 \text{Cosh}(R_1 y) -$
214 $k_1 D_2 \text{Sinh}(R_1 y) - k_1 D_3 \text{Cos}(R_2 y) - k_1 D_4 \text{Sin}(R_2 y) = 0$

215 2) $\mu_{yx} = k_2 \bar{\omega}_1$, where $\bar{\omega}_1 = \frac{1}{2} \frac{\partial w}{\partial y}$

216 This boundary condition yields the following equation in four unknown arbitrary constants
217 $D_1 L_3 \text{Cosh}(R_1 y) + D_2 L_3 \text{Sinh}(R_1 y) - D_3 L_4 \text{Cos}(R_2 y) - D_4 L_4 \text{Sin}(R_2 y) - \frac{k_2}{2} D_1 R_1 \text{Sinh}(R_1 y) -$
218 $\frac{k_2}{2} D_2 R_1 \text{Cosh}(R_1 y) + \frac{k_2}{2} D_3 R_2 \text{Sin}(R_2 y) - \frac{k_2}{2} D_4 \text{Cos}(R_2 y) = 0$

219 where σ_{yz} is force stress, μ_{yx} is couple stress, w is displacement component, $\bar{\omega}_1$ is rotation
220 vector, k_1 , k_2 are tangential and rotational spring coefficients, respectively.

221 Enforcing the above boundary conditions at the plate boundaries ($y = \pm b$) leads to the
222 following set of four homogeneous equations

$$\begin{pmatrix} D_1(L_1S_{11} - k_1C_{11}) + D_2(L_1C_{11} - k_1S_{11}) - \\ D_3(L_2S_{12} + k_1C_{12}) + D_4(L_2C_{12} - k_1S_{12}) \end{pmatrix} = 0 \quad (11)$$

223

$$\begin{pmatrix} D_1(-L_1S_{11} - k_1C_{11}) + D_2(L_1C_{11} + k_1S_{11}) - \\ D_3(-L_2S_{12} + k_1C_{12}) + D_4(L_2C_{12} + k_1S_{12}) \end{pmatrix} = 0 \quad (12)$$

224

$$\begin{pmatrix} D_1(L_3C_{11} - \frac{k_2}{2}R_1S_{11}) + D_2(L_3S_{11} - \frac{k_2}{2}R_1C_{11}) - \\ D_3(L_4C_{12} - \frac{k_2}{2}R_2S_{12}) - D_4(L_4S_{12} + \frac{k_2}{2}R_2C_{12}) \end{pmatrix} = 0 \quad (13)$$

225

$$\begin{pmatrix} D_1(L_3C_{11} + \frac{k_2}{2}R_1S_{11}) + D_2(-L_3S_{11} - \frac{k_2}{2}R_1C_{11}) - \\ D_3(L_4C_{12} + \frac{k_2}{2}R_2S_{12}) - D_4(-L_4S_{12} + \frac{k_2}{2}R_2C_{12}) \end{pmatrix} = 0 \quad (14)$$

226 here $S_{11} = \text{Sinh}(R_1b)$, $S_{12} = \text{Sin}(R_2b)$, $C_{11} = \text{Cosh}(R_1b)$, $C_{12} = \text{Cos}(R_2b)$.

227 For the Eqs. (11-14) to possess the non-trivial solution, the determinant of unknown coef-
228 ficients must become zero.

$$|d_{ij}|_{4 \times 4} = 0 \quad (15)$$

229 where $d_{11} = -k_1C_{11}$, $d_{12} = L_1C_{11}$, $d_{13} = -k_1C_{12}$, $d_{14} = L_2C_{12}$, $d_{21} = L_1S_{11}$, $d_{22} = -k_1S_{11}$, $d_{23} =$
230 $-L_2S_{12}$, $d_{24} = -k_1S_{12}$, $d_{31} = 2L_3C_{11}$, $d_{32} = -k_2R_1C_{11}$, $d_{33} = -2L_4C_{12}$, $d_{34} = -k_2R_2C_{12}$, $d_{41} =$
231 $-k_2R_1S_{11}$, $d_{42} = 2L_3S_{11}$, $d_{43} = k_2R_2S_{12}$, $d_{44} = -2L_4S_{12}$

232 Solution of Eq. (15) gives the dispersion relation written below as

$$PX^2 + QX - P = 0 \quad (16)$$

233 where $X = \frac{\tanh(R_1b)}{\tan(R_2b)}$, $P = 2k_1k_2(L_2L_4R_1 - L_1L_3R_2) + k_1^2k_2^2R_1R_2 - 4L_1L_2L_3L_4$,

234 $Q = (k_1^2 - L_1^2)(k_2^2R_2^2 + 4L_4^2) + (k_1^2 + L_1^2)(-k_2^2R_1^2 + 4L_3^2) + 8k_1^2L_3L_4 - 4k_1k_2(L_2L_3R_2 + L_1L_4R_1)$

235 The analysis of Eq. (16) leads to the derivation of two distinct dispersion relations associated
236 with SH waves

$$\frac{\tanh(R_1b)}{\tan(R_2b)} = \frac{-Q - \sqrt{Q^2 + 4P^2}}{2P} \quad (17)$$

$$\frac{\tanh(R_1b)}{\tan(R_2b)} = \frac{-Q + \sqrt{Q^2 + 4P^2}}{2P} \quad (18)$$

5 Modes of SH waves

The dispersion relation (16) comprises both symmetric and anti-symmetric components. Since, Eqs. (17) and (18) are reduced forms of Eq. (16), it is pertinent to examine which of these equations corresponds to the symmetric or anti-symmetric modes.

Let the restraint parameters in the boundary conditions be relaxed *i.e.* $k_1 = k_2 = 0$. Consequently, the boundary conditions discussed in the section (4) simplify to $\sigma_{yz} = 0$, $\mu_{yx} = 0$, corresponding to stress-free boundary conditions. Under these conditions, Eq. (15) reduces to the following two distinct dispersion relations for SH waves respectively.

$$\frac{\tanh(R_1 b)}{\tan(R_2 b)} = \frac{L_2 L_3}{L_1 L_4} \quad (19)$$

$$\frac{\tanh(R_1 b)}{\tan(R_2 b)} = \frac{-L_1 L_4}{L_2 L_3} \quad (20)$$

By imposing the symmetric conditions ($D_2 = D_4 = 0$) and anti-symmetric conditions ($D_1 = D_3 = 0$) as in (YU and YU, 1996) on Eqs. (11-14), the derived Eqs. (19) and (20) are directly obtained for the stress-free case. Accordingly, Eqs. (17) and (18) can be identified as representing the symmetric and anti-symmetric modes of SH waves.

6 The special cases

For stress-free boundary conditions, the surface tractions vanish, whereas rigid surface boundary conditions enforce zero displacements at the surface. Elastically restrained boundary conditions provide a transition between stress-free and rigid boundary conditions by introducing finite stiffness at the boundary. These three boundary conditions are discussed in this section.

Case:1 Stress-free boundary conditions

Stress free boundary conditions are achieved by specifying $k_1 = k_2 = 0$ in the boundary conditions discussed in section (4). Upon using these conditions in Eqs. (11-14), two dispersion relations (19) and (20) are attained in section (5) which are same as discussed by SHARMA and KUMAR (2023).

Case:2 Mixed boundary conditions

In the boundary conditions mentioned in the section (4), if $k_1 \rightarrow \infty$ and $k_2 = 0$, then the conditions become $w = 0$, $\mu_{yx} = 0$. Thus, the solution of the determinant in Eq. (15) produces two dispersion relations for the SH waves

$$\text{Cosh}(R_1 b) \text{Cos}(R_2 b) = 0 \quad (21)$$

$$\text{Sinh}(R_1 b) \text{Sin}(R_2 b) = 0 \quad (22)$$

Using the above mentioned mixed boundary conditions, the Eqs. (21) and (22) can also be derived from the Eqs. (17) and (18). Thus, Eqs. (21) and (22) can be classified as symmetric

266 and anti-symmetric mode of vibrations of SH waves, respectively.

267

268 **Case:3 Rigid boundary conditions**

269 In rigidly fixed boundary conditions, both displacement and rotation are totally restricted at
270 the boundary. Thus, in this case $w = 0$, $\bar{\omega}_1 = 0$, obtained by putting $k_1 \rightarrow \infty, k_2 \rightarrow \infty$ in
271 the boundary conditions stated in section (4). The determinant in Eq. (15) reduces to the
272 following two distinct dispersion relations

$$\frac{\tanh(R_1 b)}{\tan(R_2 b)} = \frac{-R_2}{R_1} \quad (23)$$

$$\frac{\tanh(R_1 b)}{\tan(R_2 b)} = \frac{R_1}{R_2} \quad (24)$$

273 On the basis of the discussion in the case 1 and case 2 regarding obtaining dispersion relations
274 directly by using these conditions in Eqs. (17) and (18), we can group Eqs. (23) and (24) to
275 as symmetric and anti-symmetric modes of vibrations of SH waves in a micro-structural plate
276 subject to rigid boundary conditions.

277 **7 Verification and comparative analysis of the model**

278 **7.1 Assessment of model consistency with classical theory**

279 Within the framework of Consistent Couple Stress Theory (CCST), stress-strain behavior of
280 a material is governed not only by the classical continuum parameters but also by an intrinsic
281 material length scale parameter (l). This quantitatively characterizes size-dependent behavior
282 resulting from the material's micro-structural effects.

283 In the vanishing length scale limit ($l \rightarrow 0$), higher order couple stress effects are suppressed.
284 The differential equation (6) reduces to the following differential equation of second order

$$\frac{d^2 h(y)}{dy^2} - \left(\xi^2 \left(1 - \frac{C^2}{C_2^2} \right) - \left(\frac{\Omega_1^2 + \Omega_2^2}{C_2^2} \right) \right) h(y) = 0 \quad (25)$$

Solution of Eq. (25) will be

$$h(y) = E_1 \cos(\beta y) + E_2 \sin(\beta y)$$

285 and thus

$$w(y) = (E_1 \cos(\beta y) + E_2 \sin(\beta y)) e^{i\xi(x-Ct)} \quad (26)$$

286 where E_1, E_2 are arbitrary constants and $\beta^2 = \frac{\Omega_1^2 + \Omega_2^2}{C_2^2} - \xi^2 \left(1 - \frac{C^2}{C_2^2} \right)$.

287 The restrained boundary condition in section (4) simplifies to $\sigma_{yz} = k_1 w$. Applying this
288 condition at the two surfaces ($y = \pm b$) of the plate, following pair of homogeneous equations
289 are obtained

$$E_1(-k_1) \cos(\beta b) + E_2(\mu\beta) \cos(\beta b) = 0 \quad (27)$$

$$E_1(\mu\beta)\sin(\beta b) - E_2(k_1)\sin(\beta b) = 0 \quad (28)$$

290 Enforcing a non-trivial solution for Eqs. (27) and (28) leads to the following equation

$$(k_1^2 - (\mu\beta)^2) \sin(\beta b)\cos(\beta b) = 0 \quad (29)$$

291 Eq. (29) is the dispersion relation for the propagation of SH waves in a micro-structured
292 rotating plate with restrained boundary condition.

293 On assuming the tangential spring coefficient (k_1) to be zero, the boundary condition be-
294 comes $\sigma_{yz} = 0$. Thus, Eq. (29) reduces to $\sin(\beta b)\cos(\beta b) = 0$, which can be further simplified
295 to the resulting pair of dispersion relations

$$\sin(\beta b) = 0, \quad i.e., \quad \beta b = \frac{n\pi}{2}, \quad n = 0, 2, 4... \quad (30)$$

$$\cos(\beta b) = 0, \quad i.e., \quad \beta b = \frac{n\pi}{2}, \quad n = 1, 3, 5... \quad (31)$$

297 Here, β is still having the rotational term in it. Thus, Eqs. (30) and (31) correspond to
298 the symmetric and anti-symmetric modes of SH wave, respectively, in a rotating plate under
299 classical theory of elasticity.

300 Furthermore, in the absence of rotation ($\Omega_1 = \Omega_2 = 0$), one obtains $\beta = \xi^2 \left(\frac{C^2}{C_2^2} - 1 \right)$. In
301 this case, Eqs. (30) and (31) represent the symmetric and anti-symmetric modes of SH wave,
302 respectively, in an elastic plate as derived in GRAFF (2012). This confirms that the proposed
303 model is consistent with classical elasticity theory and accurately reproduces the conventional
304 behavior in the absence of micro-structural effects, rotational influences and restrained coeffi-
305 cients, thereby validating the formulation.

306 7.2 Comparison with the existing studies

307 A review of the literature indicates that numerous studies have examined the propagation of
308 various elastic waves, investigating the influence of factors such as micro-structural parameter,
309 rotation, stiffness, and boundary conditions in different elastic media. However, when focus is
310 restricted to plate structures, the available research remains comparatively limited. The present
311 study investigates the combined effects of rotation, length scale parameter and elastically re-
312 strained boundary conditions (ERBC) on SH wave propagating in an elastic plate within the
313 framework of CCST theory. This section presents a comparison of the proposed model with
314 existing studies, as summarized in Table 1, along with relevant references.

Table 1: Comparison of present model with existing studies

Model	Rotation	Characteristic length	ERBC	Problem type	Relevant References
Classical Elasticity	✗	✗	✗	SH waves in plate	GRAFF(2012) KUZNETSOV(2006)
	✓	✗	✗	Reflection and refraction of plane waves	SCHOENBERG, CENSOR(1973); CENSOR, SCHOENBERG(1973)
CCST	✗	✓	✗	Lamb waves in plate	SHARMA and KUMAR(2014)
	✗	✓	✗	SH waves in coupled plates	DUA and SHARMA(2024)
	✗	✗	✓	waves in elastic media	MOUKHOMODIAROV et al (2012); MOUKHOMODIAROV and ROGERSON (2012)
	✗	✓	✗	SH waves in plate	SHARMA and KUMAR(2023)
	✓	✓	✓	Reflection and refraction of plane waves	WANG et al (2017)
	✓	✓	✓	SH waves in plate	Present study

8 Numerical results and discussion

SH waves in a plate display multimodal behavior along with dispersion characteristics. One notable aspect of this study involves the combined effect of rotation and stiffness coefficients on the phase velocity dispersion. For graphical illustration of the results, the material parameters are taken (WANG *et al.*, 2017) as $\lambda = 2.2 \times 10^{10} \text{ N/m}^2$, $\mu = 1.1 \times 10^{10} \text{ N/m}^2$, $\rho = 2.6 \times 10^3 \text{ kg/m}^3$. The standard values of stiffness constants are $k_1 = 10^{16} \text{ N/m}^3$ and $k_2 = 10^4 \text{ N/m}$.

The characteristic length is chosen to be of the same order as the plate thickness $l \sim b$ to capture size-dependent effects. The stiffness parameters range from zero (stress-free) to infinity (perfectly constrained); however, finite values are adopted to represent realistic surface conditions in line with established literature. The rotational speed (10^8 rad/sec) though high for macroscopic systems, is appropriate for micro/nano-scale devices and enables capturing rotational effects on wave propagation. Moreover, to address this, dimensionless parameters have also been introduced. Since, similar behavior is observed in both symmetric and anti-symmetric modes, the graphs are presented only for the symmetric cases.

8.1 Symmetric and anti-symmetric modes of SH waves

The dispersion relation expressed in Eq. (16), derived in section (4) reduces to two distinct forms as presented in Eqs. (17) and (18). These two relations correspond respectively to the symmetric and anti-symmetric cases of propagation, consistent with the classification outlined in section (5). Figures (1) and (2) depict the variation of non dimensional phase velocity with respect to the non dimensional wavenumber for both cases, highlighting the influence of the rotational parameter. These figures plot the non dimensional phase velocity against non dimensional wavenumber for mode numbers, $n = 1, 2, 3$. These profiles clearly show two robust features. First, for both the non-rotating and rotating cases the phase-velocity curves for successive mode numbers are shifted upward (higher modes propagate with higher phase velocity for a given non dimensional wave number), indicating the classical higher-mode stiffening. Second, the introduction of rotation systematically reduces the phase velocity of every mode. This reflects the additional effective inertia introduced by rotation (rotational inertia and coriolis-type coupling) mitigates the stiffening effect associated with couple stress. A modal asymmetry is also evident: the symmetric branch does not admit a fundamental (low-frequency) propagating mode under the present couple-stress boundary conditions, whereas the anti-symmetric branch supports a fundamental mode that exhibits a trend opposite to that of the higher anti-symmetric branches and is characterized by a cutoff frequency/wavenumber, below which wave propagation does not occur. The cutoff and the reversed low-frequency trend are controlled by the non-dimensional characteristic length and the rotational parameter, demonstrating a sensitive competition between microstructural stiffening and rotational softening.

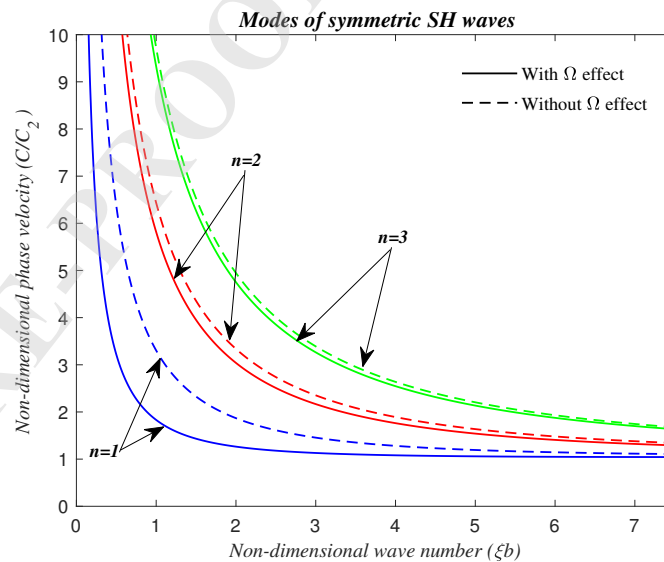


Figure 1: Phase velocity profiles for illustrating modes of symmetric SH waves

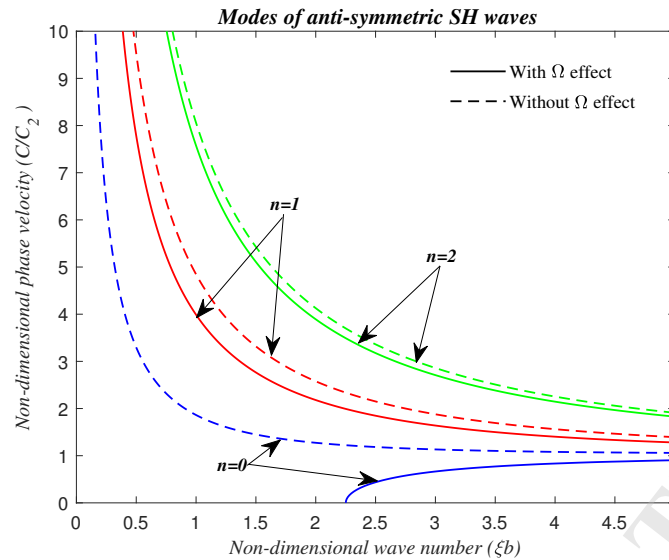


Figure 2: Phase velocity profiles for illustrating modes of anti-symmetric SH waves

350 8.2 Effect of dimensionless rotation parameter (Ω_2^*)

351 In a rotating couple-stress elastic medium, the propagation of SH waves is strongly influ-
 352 enced by the inertial effects associated with the rotational components, which directly modify
 353 the governing equations of motion and thereby alter the dispersion characteristics. The two
 354 components of the rotation vector, Ω_1 and Ω_2 , appear in the governing relations through their
 355 sum of squares, $\Omega_1^2 + \Omega_2^2$, as evident from the values of R_1 and R_2 in section-3

356 Dimensionless parameters play a crucial role in characterizing wave propagation behavior
 357 by encapsulating the combined effects of material properties and characteristic length scales. In
 358 particular, $\Omega_2^* = (\Omega_2 l)/C_2$ quantifies the relative influence of characteristic length, rotation and
 359 shear wave velocity. Accordingly, the dispersion behavior of symmetric and anti-symmetric
 360 SH waves is examined under varying values of Ω_2^* . Figure (3) shows that for a fixed wave
 361 number (ξb), an increase in the value of Ω_2^* leads to the systematic reduction in phase velocity.
 362 This effect is more pronounced at smaller wave numbers. The significant variation in phase
 363 velocity at small wave numbers reflects strong dispersive effects that progressively diminish as
 364 (ξb) increases. At large wave numbers, the dispersion curves coalesce due to the diminishing
 365 influence of coupling and micro-structural effects.

366 Since, Ω_1 and Ω_2 enter the governing equations through the combined quadratic form $\Omega_1^2 +$
 367 Ω_2^2 ; consequently, dimensionless rotation parameter Ω_1^* contributes identically to the phase
 368 velocity of SH waves.

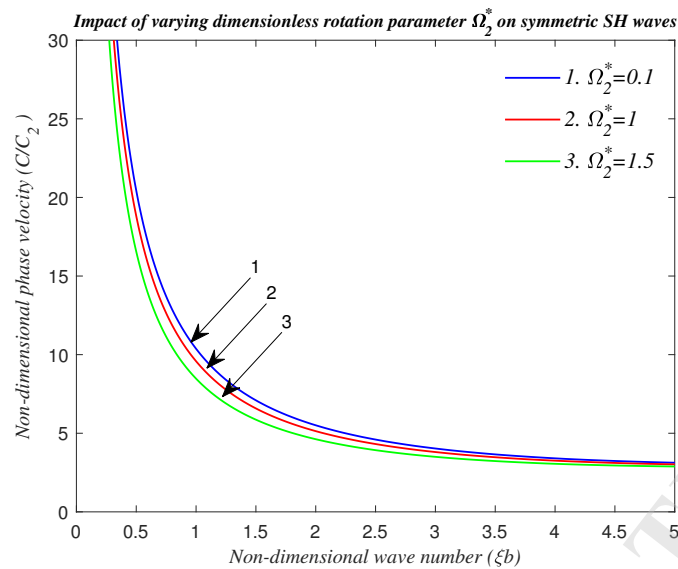


Figure 3: Phase velocity profiles for illustrating impact of dimensionless rotation parameter Ω_2^* on symmetric SH waves

369 8.3 Effects of plate thickness

370 Geometric parameters play a crucial role in determining the propagation characteristics of
 371 waves in solid media. In the present case, the finite thickness of the plate governs the boundary
 372 conditions and, consequently, the modal structure of the SH waves. Figure (4) illustrates the
 373 variation of non dimensional phase velocity with respect to non- dimensional wavenumber for
 374 different plate thicknesses.

375 It is observed that the phase velocity exhibits a negative correlation with plate thickness,
 376 whereby as the thickness increases, the phase velocity decreases. Both the non-rotating and
 377 rotating cases display this decreasing trend. However, when rotation is incorporated, the phase
 378 velocity profiles are slightly suppressed. Overall, the results emphasize that geometric scaling,
 379 in conjunction with rotational dynamics and couple stress effects, plays a decisive role in defin-
 380 ing the dispersive response of SH waves in finite plates. Increasing plate thickness tends to
 381 reduce the phase velocity. Rotational effects further enhance this trend, leading to additional
 382 suppression of wave propagation.

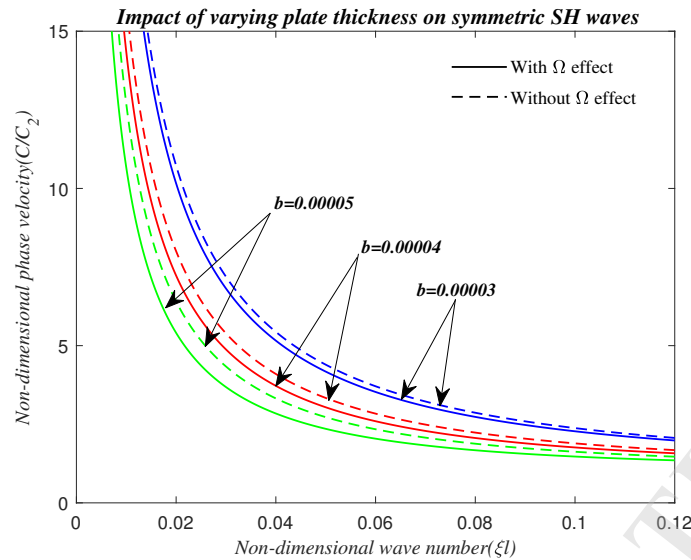


Figure 4: Phase velocity profiles for illustrating impact of varying plate thickness on symmetric SH waves

383 8.4 Effects of characteristic length parameter (l)

384 The characteristic length parameter (l) is a fundamental micro-structural quantity in couple
 385 stress elasticity, representing the internal material length scale or the average size of the mi-
 386 crostructural cells. Typically of the order of $10^{-6} m$, this parameter governs the extent to which
 387 micro-rotations and couple stresses influence the deformation field. To examine its impact on
 388 SH wave propagation under constrained plate conditions, phase velocity profiles were computed
 389 for different values of l . As shown in figure (5), an increase in the characteristic length param-
 390 eter leads to a significant enhancement in phase velocity. This behavior can be attributed to
 391 the strengthening effect introduced by the microstructural interactions, which effectively in-
 392 crease the material's resistance to shear deformation. As l increases, the contribution of couple
 393 stresses to the total stress field becomes more dominant, resulting in an apparent stiffening of
 394 the medium and hence higher phase velocities. The trend clearly indicates that the internal
 395 length parameter acts as a stiffening factor, counteracting the dispersive softening associated
 396 with geometric and rotational effects. Consistent with earlier observations, the inclusion of
 397 rotational parameters slightly reduces the overall magnitude of phase velocity for each value
 398 of l . This demonstrates a competing interplay between microstructural stiffening (through l)
 399 and rotational softening (through Ω_1, Ω_2). At higher values of l , however, the microstructural
 400 effects tend to dominate, leading to comparatively higher phase velocity even in the presence
 401 of rotation.

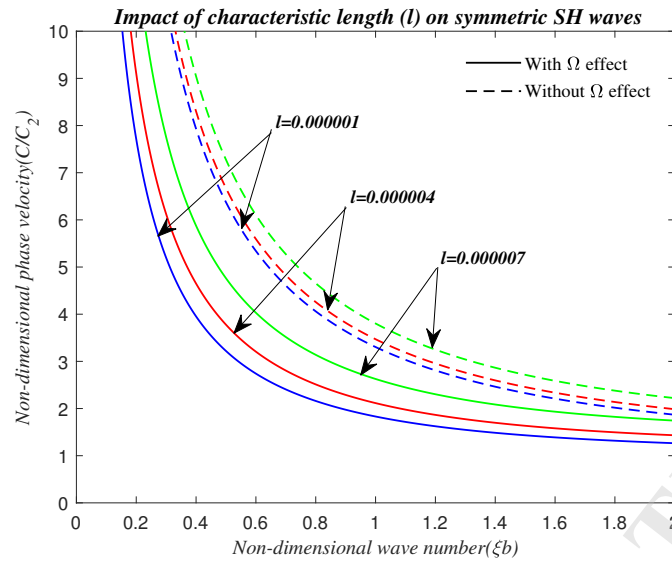


Figure 5: Phase velocity profiles for illustrating impact of varying characteristic length on symmetric SH waves

402 8.5 Effect of dimensionless parameters (l/b)

403 By analyzing the pronounced effects of the characteristic length scale and plate thickness on
 404 the dispersion behavior, the phase velocity is examined under rotational effects as a function
 405 of the dimensionless ratio l/b in figure (6). It is observed that increasing this ratio leads to
 406 a notable amplification of phase velocity, reflecting the enhanced resistance of the material to
 407 shear deformation. This effect is due to the additional stiffness contributed by microstructural
 408 effects. Proper consideration of the length-to-thickness ratio is therefore essential for accurately
 409 predicting deformation, natural frequencies, and stress distribution in microstructured or thin-
 410 scale structures.

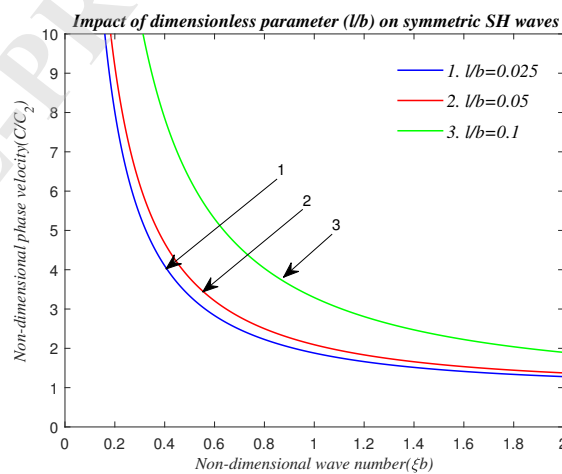


Figure 6: Phase velocity profiles for illustrating impact of dimensionless parameter l/b on symmetric SH waves

411 8.6 Effect of dimensionless stiffness parameters (k_1^* and k_2^*)

412 Wave propagation in microstructured solids are governed by both, the geometrical con-
 413 figuration and intrinsic physical properties. The introduction of dimensionless parameters
 414 $k_1^* = (k_1 l)/\mu$ and $k_2^* = k_2/(l\mu)$ consolidate the effects of material constant (μ), stiffness co-
 415 efficients (k_1 and k_2) and the length scale parameter (l).

416 Figure (7) indicates that increasing k_1^* enhances the contribution of the material's mi-
 417 crostructure, thereby increasing the effective shear stiffness of the plate and, consequently,
 418 the phase velocity. At constant wave number, profiles exhibit a direct relationship between the
 419 phase velocity and the dimensionless stiffness parameter, k_1^* . A pronounced dispersive effect
 420 is observed, particularly at short wavelengths, where the phase velocity exhibits a more rapid
 421 increase.

422 Contrary to the effect of k_1^* , the dimensionless quantity k_2^* in figure (8) is inversely related
 423 to the phase velocity. Higher values of k_2^* enhances the system's resistance to dynamic defor-
 424 mation and strongly constrain microrotations, consequently oppose wave propagation despite
 the apparent increase in boundary stiffness.

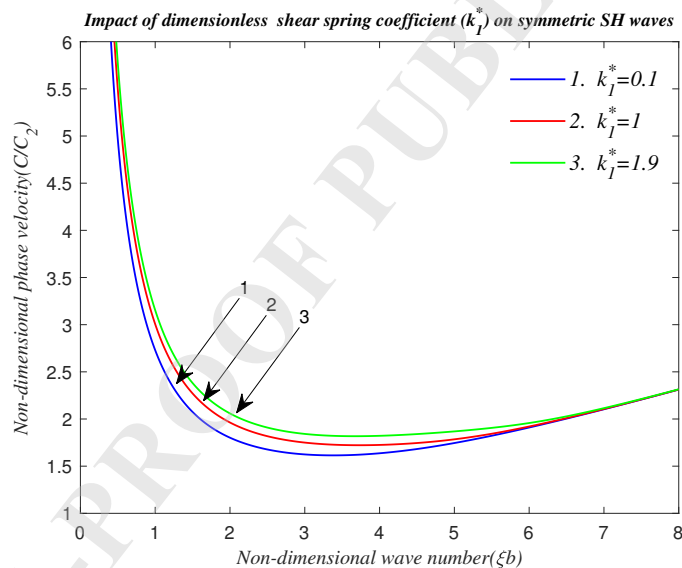


Figure 7: Phase velocity profiles for illustrating impact of dimensionless shear stiffness parameter (k_1^*) on symmetric SH waves

425

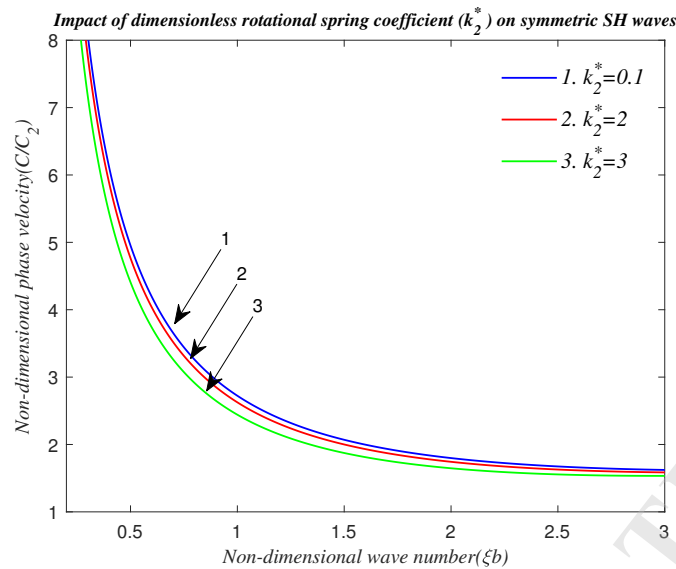


Figure 8: Phase velocity profiles for illustrating impact of dimensionless rotational stiffness parameter (k_2^*) on symmetric SH waves

426 8.7 Effect of boundary conditions

427 The dispersion relations labeled as (19 – 20), (21 – 22), and (23 – 24) correspond to three
 428 distinct types of boundary conditions: stress-free, mixed, and fixed, respectively. These rela-
 429 tions have been derived under both rotating and non-rotating conditions and are graphically
 430 represented in figures (9) and (10) for both symmetric and anti-symmetric shear-horizontal
 431 (SH) wave modes, respectively.

432 The stress-free boundary condition implies that both the normal and shear stress compo-
 433 nents at the boundary are zero, representing a completely traction-free surface (ideal for free
 434 surfaces like air-solid interfaces). The fixed boundary condition assumes zero displacement at
 435 the surface, meaning the boundary is perfectly clamped or rigidly constrained, often used to
 436 model bonding with rigid substrates. The mixed boundary condition represents an interme-
 437 diate case where one stress component and one displacement component are set to zero. For
 438 example, tangential stress might be zero while normal displacement is constrained, modeling
 439 partial bonding or imperfect adhesion.

440 From the dispersion curves in figure (9), it is observed that for the symmetric mode of
 441 SH waves, the phase velocity corresponding to the stress-free boundary condition lies between
 442 those obtained for the mixed and fixed boundary conditions. In contrast, for the anti-symmetric
 443 mode shown in figure (10), a different ordering of phase velocity profiles is evident. In this case,
 444 the phase velocity is highest for the stress-free boundary condition and lowest for the mixed
 445 condition. Overall, the results indicate that the influence of boundary conditions on phase
 446 velocity is mode-dependent, leading to distinct trends for symmetric and anti-symmetric wave
 447 propagation.

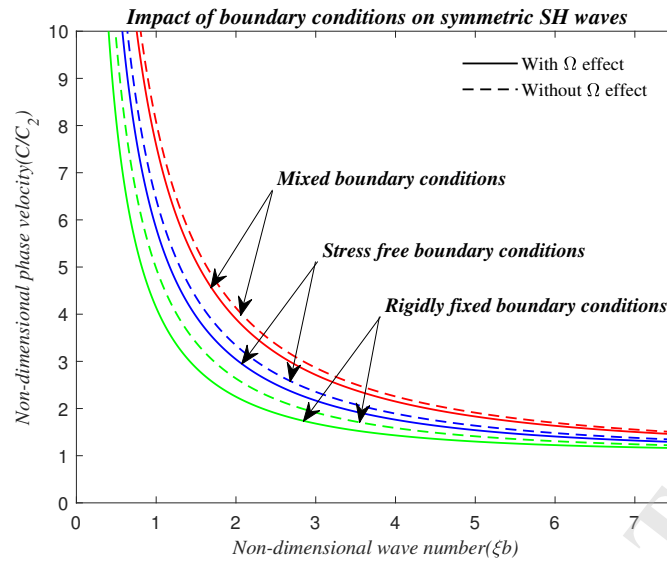


Figure 9: Phase velocity profiles for illustrating impact of boundary conditions on symmetric SH waves

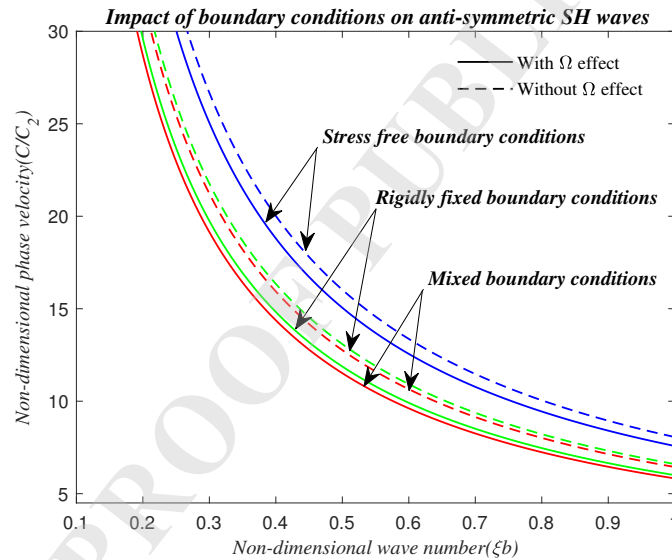


Figure 10: Phase velocity profiles for illustrating impact of boundary conditions on anti-symmetric SH waves

448 8.8 SH-wave propagation under different modeling assumptions

449 SH wave propagation in a micro-structured plate is governed by size-dependent effects,
 450 rotational motion, and boundary constraints, and the present model captures their combined
 451 influence. In the absence of rotation and boundary constraints, the formulation reduces to the
 452 CCST model, while further neglect of microstructural effects recovers the classical continuum
 453 model. The results show that the incorporation of the characteristic length parameter within
 454 the classical framework leads to an increase in phase velocity, as illustrated by curves 1 (present
 455 model), 2 (CCST), and 3 (classical) in figure (11). However, the inclusion of rotational effects
 456 reduce the phase velocity, resulting in the present model exhibiting an intermediate response
 457 between the classical and CCST predictions.

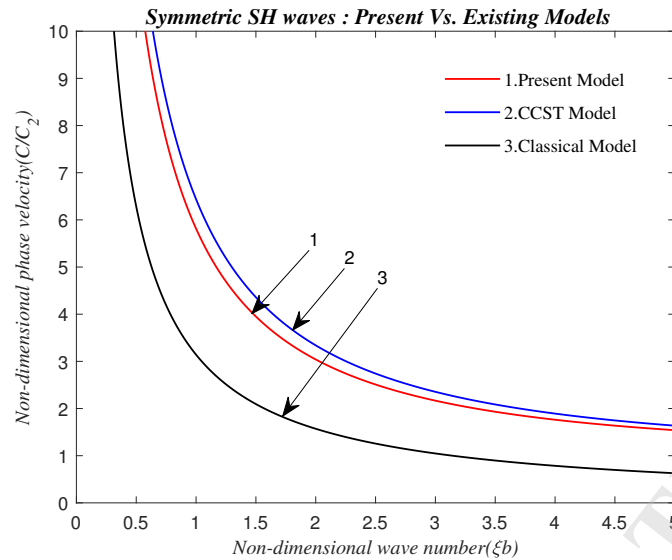


Figure 11: Phase velocity profiles for illustrating symmetric SH waves behavior under different models

9 Conclusion

The present study has thoroughly examined the dispersion behavior of shear-horizontal (SH) waves in a micro-structured couple stress elastic plate subjected to rotational effects induced by coriolis and centrifugal forces. The primary objective was to investigate the role of restrained boundary conditions, micro-structural and geometric features such as characteristic length and plate thickness as well as the dimensionless parameters on the wave propagation characteristics.

Plate rotation effects on SH wave modes: A detailed comparison between cases with and without plate rotation reveals that rotation leads to a consistent reduction in phase velocity across both symmetric and skew-symmetric SH wave modes. While symmetric modes exhibit minimal sensitivity, anti-symmetric modes show a more pronounced effect. Interestingly, the fundamental mode in skew-symmetric profiles undergoes a polarity reversal and exhibits a cut-off frequency, a distinct behavior not seen in non-rotating plates.

Role of dimensionless parameters, (Ω_1^* and Ω_2^*): The analysis indicates that both the rotation parameters (Ω_1^*) and (Ω_2^*) influence wave dispersion symmetrically due to the appearance of Ω_1 and Ω_2 as additive and squared terms in the governing equations. An increase in either dimensionless parameter leads to a decrease in phase velocity. This reduction is consistently observed in both symmetric and anti-symmetric SH wave modes.

Effect of plate thickness: The study reveals a clear inverse relationship between plate thickness and phase velocity, showing that an increase in thickness leads to a consistent reduction in phase velocity. This behavior highlights the sensitivity of wave propagation to geometric parameters, with plates of greater thickness exhibiting comparatively slower wave transmission.

482

483 **Impact of characteristic length parameter (l):** As a core parameter of couple stress
484 theory, the characteristic length significantly influences wave behavior. Increasing ' l ' enhances
485 phase velocity by amplifying micro-structural effects, thus enabling more efficient wave energy
486 transmission. This effect highlights the importance of considering material length scales in
487 micro-structured wave-guide designs.

488

489 **Effect of dimensionless quantity, (l/b):** The results reveals that the phase velocity
490 pattern is significantly influenced by variations in the ratio l/b . Changes in this parameter
491 lead to noticeable modifications in the propagation characteristics of SH waves, highlighting
492 the sensitivity of phase velocity to geometric scaling. An increase in l/b results in an amplifi-
493 cation of phase velocity and enhanced dispersive behavior. This suggests that the structural
494 configuration plays a crucial role in governing wave dynamics and is essential for optimizing
495 design parameters in applications involving guided wave propagation.

496

497 **Impact of dimensionless parameters, (k_1^* and k_2^*):** The shear stiffness parameter (k_1)
498 governs resistance to tangential deformation, whereas the rotational stiffness (k_2) regulates an-
499 gular restraint. The phase velocity exhibits contrasting dependence on the non-dimensional
500 parameters (k_1^*) and (k_2^*) Specifically, an increase in (k_1^*) leads to an increase in phase velocity,
501 attributable to the enhanced stiffness that facilitates faster wave propagation. In contrast,
502 an increase in (k_2^*) is associated with a reduction in phase velocity. Thus, these parameters
503 influence wave speed in distinct ways and may be tuned to achieve desired propagation charac-
504 teristics, as is often explored in wave-control technologies where analogous stiffness parameters
505 are adjusted to tailor wave speed, attenuation, or confinement.

506

507 **Implications of boundary conditions:** The study also provides a comparative analysis
508 of three different boundary condition types—stress-free, fixed, and mixed. By examining all
509 three cases, the study highlights how boundary constraints can modify phase velocity profiles
510 and wave behavior, providing insights into the sensitivity of wave propagation to edge condi-
511 tions.

512

513 **Assessment of SH wave propagation across different models:** SH wave propaga-
514 tion in a microstructured plate is influenced by size-dependent effects, rotational motion, and
515 boundary conditions, all of which are incorporated in the present model. The result shows that
516 introducing a characteristic length scale increases phase velocity, while rotation decreases it,
517 resulting in phase velocity curve of the present model to lie between classical theory and CCST
518 predictions.

519 This analysis highlights the nuanced interplay between microstructure, boundary conditions,
520 rotation, and geometric parameters in influencing the behavior of SH waves. The results con-
521 tribute to a broader theoretical understanding of wave propagation in micro-structured rotating

522 plates and may offer useful guidance for design considerations in certain related engineering
523 applications.

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527 CONFLICT OF INTEREST

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530 AUTHORS' CONTRIBUTIONS

531 All authors have contributed equally in the preparation of different sections of the manuscript.

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